

# **Venturix**

Version 2.0

## VENTURIMETER THEORY for INCOMPRESSIBLE & COMPRESSIBLE FLOW

by  
Mech. Res. Eng. YILMAZ YÖRÜ

Mechanical Engineering Department  
of  
OSMANGAZI UNIVERSITY  
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**A) FLOW RATE for INCOMPRESSIBLE FLOW**

For incompressible flow from Bernoulli Equation;

$$\frac{P_1}{\gamma} + \frac{V_1^2}{2g} + Z_1 = \frac{P_2}{\gamma} + \frac{V_2^2}{2g} + Z_2$$

$$Z_1 = Z_2$$

$$Q = A_1.V_1 = A_2.V_2 \Rightarrow V_1 = \frac{Q}{A_1} ; V_2 = \frac{Q}{A_2}$$

$$\frac{Q^2}{2g.A_1^2} - \frac{Q^2}{2g.A_2^2} = \frac{P_2}{\gamma} - \frac{P_1}{\gamma}$$

$$\frac{Q^2}{2g.A_1^2} - \frac{Q^2}{2g.A_2^2} = \frac{P_2}{\gamma} - \frac{P_1}{\gamma} \quad h_1 = \frac{P_1}{\gamma} ; h_2 = \frac{P_2}{\gamma}$$

$$\frac{Q^2}{2g} \left( \frac{1}{A_1^2} - \frac{1}{A_2^2} \right) = h_2 - h_1$$

$$Q^2 = \frac{2g(h_2 - h_1)}{\left( \frac{1}{A_1^2} - \frac{1}{A_2^2} \right)}$$

$$Q = A_2 \sqrt{\frac{2g(h_2 - h_1)}{\left( 1 - \left( \frac{d_1}{d_2} \right)^4 \right)}} \quad \text{Flow Rate For Incompressible Flow (m}^3\text{/sec)}$$

$$Q^* = C_d . A_2 \sqrt{\frac{2g(h_2 - h_1)}{\left( 1 - \left( \frac{d_1}{d_2} \right)^4 \right)}} \quad \text{Real Flow Rate For Incompressible Flow (m}^3\text{/sec)}$$

$$\beta = \frac{D_2}{D_1}$$

Long Radius Nozzle

$$C = C_d \approx 0.9965 - 0.00653(\beta)^{1/2} \left( \frac{10^6}{\text{Re}_{D_1}} \right)^{1/2}$$

Short Radius ISA 1932 Flow Nozzle

$$C = C_d = 0.9900 - 2262\beta^{4.1} + (0.000215 - 0.001125\beta + 0.00249\beta^{4.7}) \left( \frac{10^6}{\text{Re}_D} \right)^{1.15}$$

Venturimeter

$$C = C_d \approx 0.9858 - 0.196\beta^{4.5}$$

**B) FLOW RATE for COMPRESSIBLE FLOW**

Energy Equation for Two Point;

$$\frac{U_2^2 - U_1^2}{2} = h_1 - h_2$$

$$U_2^2 - U_1^2 = 2(h_1 - h_2)$$

$$U_2^2 - U_1^2 = 2c_p T_1 \left( 1 - \frac{T_2}{T_1} \right)$$

$$\frac{T_2}{T_1} = \left( \frac{P_2}{P_1} \right)^{\frac{k-1}{k}}$$

$$\frac{\rho_2}{\rho_1} = \left( \frac{P_2}{P_1} \right)^{\frac{1}{k}}$$

$$U_2^2 \left( 1 - \left( \frac{U_1}{U_2} \right)^2 \right) = 2c_p T_1 \left( 1 - \left( \frac{P_2}{P_1} \right)^{\frac{k-1}{k}} \right)$$

$$\rho_1 U_1 A_1 = \rho_2 U_2 A_2$$

$$\frac{U_1}{U_2} = \frac{\rho_2 A_2}{\rho_1 A_1} = \left( \frac{P_2}{P_1} \right)^{\frac{1}{k}} \left( \frac{A_2}{A_1} \right)$$

$$U_2^2 = \frac{2c_p T_1 \left( 1 - \left( \frac{P_2}{P_1} \right)^{\frac{k-1}{k}} \right)}{1 - \left( \frac{A_2}{A_1} \right)^2 \left( \frac{P_2}{P_1} \right)^{\frac{2}{k}}}$$

$$U_2 = \sqrt{\frac{\frac{2kRT_1}{k-1} \left( 1 - \left( \frac{P_2}{P_1} \right)^{\frac{k-1}{k}} \right)}{1 - \left( \frac{A_2}{A_1} \right)^2 \left( \frac{P_2}{P_1} \right)^{\frac{2}{k}}}}$$

Velocity at Throat (m/sec)

$$Q = C.A_2.U_2$$

$$Q = C.A_2 \sqrt{\frac{\frac{2kRT_1}{k-1} \left( 1 - \left( \frac{P_2}{P_1} \right)^{\frac{k-1}{k}} \right)}{1 - \left( \frac{A_2}{A_1} \right)^2 \left( \frac{P_2}{P_1} \right)^{\frac{2}{k}}}}$$

Real Flow Rate For Compressible Flow (m<sup>3</sup>/sec);**Resources:**

Theory and Problems of Fluid Mechanics and Hydraulics, Ranald V. GILES  
 Mechanics of Fluids, Merle C. POTTER, David C. WIGGERT, Midhat HONDZO  
 Fluid Mechanics, Frank M. WHITE,  
 Hidrolik Makinalar Laboratuar Deneyleri, Yasar PANCAR